

IN THE SPECIFICATION:

On page 1, immediately after the title, please insert the following paragraph and heading as follows:

This specification for the instant application should be granted the priority date of August 13, 2003, the filing date of the corresponding German patent application 103 37 376.4 as well as the priority date of 29 June 2004, the filing date of the corresponding International patent application PCT/EP2004/007035.

Background of the Invention.

On page 1, lines 6-7, please amend this paragraph as follows:

The present invention relates to a retroactive device ~~having the features of the preamble of Claim 1~~ as part of a hydraulic servo-steering system that also includes a hydraulic servo-valve device, in particular for motor vehicles.

On page 3, line 7, please insert the following heading:

--Summary of the Invention--

On page 3, lines 8-9, please amend this paragraph as follows:

This object is achieved by a retroactive device ~~having the features of Claim 1~~ means for hydraulically producing a restoring moment into a central position that produces the restoring moment as a function of the pressure differential between a pressure side and a low-pressure side; the hydraulic servo-steering system further has at least one valve means that, when pressure is applied in the central position of the servo-valve, is adapted to produce a pressure differential between the external chamber and the internal chamber of the valve, wherein the at least one valve means is disposed hydraulically in series with the retroactive device.

On page 4, line 11, please insert the following heading:

--Brief Description of the Drawings--

On page 5, line 10, please insert the following heading:

--Description of Specific Embodiments--.

On page 8, lines 12-21, please amend this paragraph as follows:

In this arrangement, as described above, the hydraulic basic load is produced in that the primary pressure of the hydraulic pump ~~37~~30 is applied to the retroactive elements 12, and the degree of said pressure is limited by the valve 13. The cut-off control slide 38, comprising its throttling port 39 and the cut-off valve 40, ensures that the hydraulic pressure acting on the retroactive elements 12 ceases to be applied if the pressure of the hydraulic pump 30 becomes too great. The cut-off pressure may be selected by means of the spring constant of the cut-off valve 40 and the size of the throttling port 39. The proportional valve 41 is capable of electrically controlling the extent of the restoring force.

On page 10, lines 6 – 18, please amend this paragraph as follows:

Fig. 6 illustrates a further improved embodiment. In this case, the valve 13 is configured as an electrically pilot-controlled pressure control valve. An electrically activatable proportional valve 47 adjusts the primary pressure entering the hydraulic line 37 from the pump 30, which pressure acts on the control side of the pilot valve 46. The pressure in the hydraulic line 37, and therefore before the valve device 38, may thus be controlled via the electrical proportional valve ~~36~~47. The restoring force of the servo-steering system may thus be controlled in accordance with the operating state of the motor vehicle. In a corresponding configuration of the electronics (not shown), the degree of the restoring forces may also be selected by the driver. The “driving feel” that a correspondingly configured servo-steering system imparts to the driver is then adjustable and selectable.

On page 11, after line 15, please insert the following two new paragraphs:

--The specification incorporates by reference the disclosure of German priority document 103 37 376.4 filed August 13, 2003 and PCT/EP2004/007035 filed 29 June 2004.

The present invention is, of course, in no way restricted to the specific disclosure of the

specification and drawings, but also encompasses any modifications within the scope of the appended claims.--

In addition, please add the attached abstract to the specification: